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L a o r a o o f e o t e c h a n d u c e r g r o u n d  
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... s a l v a d e d r e g e -  
f n i e e m e n f w a i e t h w o r l d p o s s e s o l  
c o m p u t i n g i n d e x t e n s i i m e l p e e r d f o r  
a n d i t h a s a l a g r e u m e r f d i f f e r e n t s e l e m  
m o d e h a e r i n d e a h d a n d y t p i r c o c e s p e c  
i t g s o o n l i n e a r c h i t a y i n c a d y s f i u s c t i t d r  
w o r l d a i l l e g y T h e e l a s t i c e f f e c t  
p r e m i a e t s i n s t e m l i n e d i m m e d e a t e f o r n i d  
s e r i e s - c o n a h e l i t w a s n e l a s t ( h o o k e  
b o d y ) d v i s c o l u s m e n t d o n y f o r e x a m p l e  
M a x w e l l m o d e l s n e l a s t i c i n s e r i e s a h  
v i s c o l u s m e n t ( 0 ) t h K e l v i n m o d e l s n e l a s t i c  
e l e m e n t a d l w e i t a l v i s c o l u s m e n t e d r t h e  
m o d e M m o d e l d K m o d e l h p p e c r o n v e i t e l s r e e  
p a r a m e t e r s m o d e l ) , T h o m s o n  
m o d e h d f o u r a n e t M r K ( B u r g e n o ' d e i l t o  
p r o s y r f e r o n f e l a s i h e m o d u l w i s c h e  
c a l l e d A B A Q S d i r e c t l y o f t h e l i s o f

Abstract—There \* # \* \* % ! \* " \* ! \* " \* \*  
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Key words—differentiability, transform,

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be expressed as: the Laplace transform (the initial conditions).

$$\text{where } \alpha, \beta = \frac{p_1 \pm \sqrt{p_1^2 - 4p_2}}{2p_2}$$

$$\bar{P}_{(s)} \bar{\sigma} = \bar{Q}_{(s)} \bar{\varepsilon} \quad (2)$$

In order to obtain the Laplace transform of the constant strain  $\varepsilon_0$  in the Laplace transform of  $1/s$ .

$$\text{three Laplace transforms of } \left[ \frac{1}{s} \right] \text{ is } \left[ \frac{1}{s} \right] \text{ so } + 6)9) \#)'$$

The Laplace inverse of the above expression,

$$Y(t) = \frac{1}{2} \left[ \alpha \left( \frac{q_1}{2} \right) \cdot e^{-(\frac{q_1}{2} + \beta)t} + \beta \cdot e^{-\beta t} \right] \quad (9)$$

The above equation is the ABQS and engineering stress-strain relation (9)

$$\sigma = M \left[ \frac{1}{2} \left( \alpha \left( \frac{q_1}{2} \right) \cdot e^{-(\frac{q_1}{2} + \beta)t} + \beta \cdot e^{-\beta t} \right) \right] \quad (10)$$

The above equation is expressed in series form.

$$G(t) = G_\infty + \sum_{i=1}^n G_i \cdot e^{-t/\tau_i} \quad (1)$$

By (1), the series is expanded with terms:

$$G_i = G_\infty + G_i \cdot e^{-t/\tau_i} \quad (2)$$

Compare (2) and (1), so

$$G_i = G_\infty + G_i \cdot e^{-t/\tau_i}$$

$$G_i = G_\infty + G_i \cdot e^{-t/\tau_i}$$

B. Conversion of the power function of the modulus

Burgers model is derived as follows which shows...

$$G(t) = G_\infty + \sum_{i=1}^n G_i \cdot e^{-t/\tau_i}$$

Fig. 1 Burgers model

The constitutive equation of Burgers model:

$$\sigma + p_1 \dot{\sigma} + p_2 \ddot{\sigma} = q_1 \varepsilon + q_2 \dot{\varepsilon} \quad (3)$$

As  $p_1, p_2, q_1, q_2$  are expressed in terms of modulus, then

$$\sigma = \frac{E_M}{E_M + E_K} \varepsilon + \frac{E_M E_K}{E_M + E_K} \dot{\varepsilon} \quad (5)$$

$$\sigma = \frac{E_M}{E_M + E_K} \varepsilon + \frac{E_M E_K}{E_M + E_K} \dot{\varepsilon}$$

Take the modulus in the three dimensions and in the normal stress in ABQS that is, for  $M, K, E_M, E_K, G_M, G_K, \tau_1, \tau_2, \mu_M, \mu_K$  so

$$p_1 = \frac{\eta'_M G_M + \eta'_M G_K + \eta'_K G_M}{G_M G_K}, \quad p_2 = \frac{\eta'_M \eta'_K}{G_M G_K} \quad (6)$$

$$q_1 = 2\eta'_M, \quad q_2 = 2 \frac{\eta'_M \eta'_K}{G_K} \quad (13)$$

Where:

$$G_M = \frac{E_M}{2(1 + \mu_M)}, \quad G_K = \frac{E_K}{2(1 + \mu_K)} \quad (7)$$

$$\tau_1 = \frac{1}{2} \frac{1}{\eta'_M}, \quad \tau_2 = \frac{1}{2} \frac{1}{\eta'_K}$$

$E_M, \mu_M, \tau_1, \tau_2$  are parameters of Burgers' model

When the constitutive equation of Burgers model is used in the case of a Mercator model

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5 a Mercator model

One-dimensional constitutive equation

Conversion of the Laplace transform,

$$\bar{\sigma} = \frac{q_1 s + q_2}{s^2 + p_1 s + p_2} \bar{\varepsilon} = \frac{1}{p_2} \left[ \frac{q_1}{(s - \alpha)} + \frac{q_1}{\beta(\alpha - \beta)} \right]$$

$$+ \left[ \frac{q_2 \alpha}{(s - \alpha)(\beta \alpha)} + \frac{q_2 \beta}{(s - \beta)(-\alpha \beta)} \right]$$

(8)

$$\frac{1}{s^2 + p_1 s + p_2}$$

$$\frac{1}{s^2 + p_1 s + p_2}$$

Fig. 2 Mercator model



stress analysis and numerical solutions. The model is based on the finite element method and is used to analyze the behavior of the structure under various loading conditions. The results show that the structure is able to withstand the applied loads without any significant deformation or failure.

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E+ EI
AE, KI
w AE, NI
EI SE+ PI
kx AE/ UI
#EI EE, WI
ii SE+ VI
AE, I
SE+ ai
EI EI II GEI I KEI KEI KI KGEI
Ffd l]x
    
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Fig. C5 Comparison of analytical and numerical results

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E+ EI
AEW EA KE
K EA KE
U X AEWA KE
t X AEWA KE
c AEWA KE
i AEFA KE
SPWA KE
SUEA KE
9: SUWA KE
W X SWEA KE
c WA KE
S EA KE
EI EI II GEI I KEI KEI KI KGEI
Ffd l]x
    
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Fig. C6 Comparison of analytical and numerical results

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The pile cap is a rigid frame foundation. The pile cap is 85m thick and the pile length is 46.5m. The pile cap is supported by four piles. The pile cap is analyzed using the finite element method. The results show that the pile cap is able to withstand the applied loads without any significant deformation or failure.

	g				
		Fine Gravelly sand			
<sup>-9</sup>	(MPa)	30	150.0	2000032500	
		0.3	0.25	0.20	0.20
<sup>10</sup>	(MP.s)	1.38	-	-	-
<sup>γ</sup>	(kN/m <sup>3</sup> )	9.20	11.0	16.0	15.0

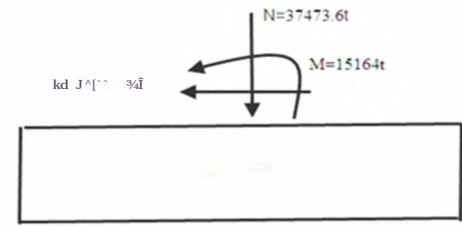


Fig. G7 Analytical results

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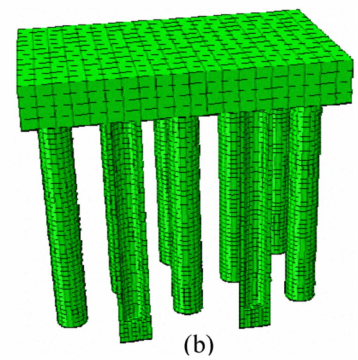


Fig. M8 shows finite element analysis of pile cap and caisson. The figure shows the finite element mesh of the pile cap and caisson. The pile cap is analyzed using the finite element method. The results show that the pile cap is able to withstand the applied loads without any significant deformation or failure.

